

5. SHS KT-joint (K)

The design of axial resistance of the SHS KT-joint is performed according to prEN 1993-1-8(2021), see Figure 5.1. The exact description and material properties of each member are given. The classification of hollow section truss-type joints as T, Y, X or K gap joints (which includes N-type joints) should be based on the method of force transfer in a joint but not on its physical appearance. Since there is no internal force in member 3, the KT joint considers as the K-joint.

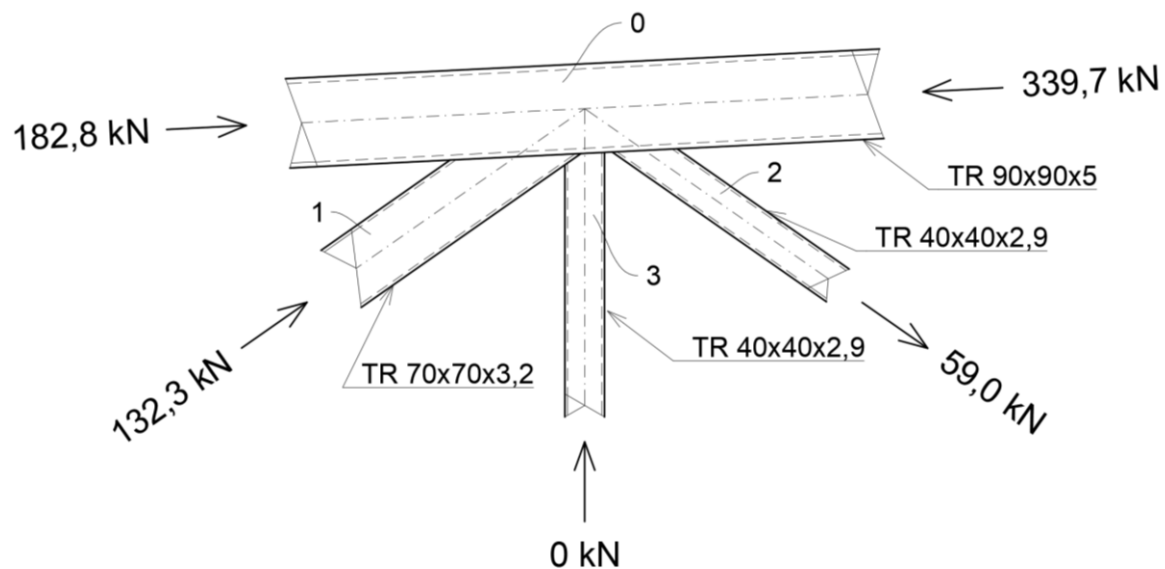


Figure 5.1: Joint geometry, internal forces

5.1. Symbols

b_i	overall out-of-plane width of RHS member i ($i=0,1,2$ or 3)
h_i	overall in-plane depth of the cross-section of member i ($i=0,1,2$ or 3)
e	eccentricity of a joint
$f_{y,i}$	yield strength of member i ($i=0,1,2$ or 3)
A_i	cross sectional area of member i ($i=0,1,2$ or 3)
$W_{el,i}$	elastic section modulus of member i ($i=0,1,2$ or 3)
θ_i	included angle between brace member i and the chord ($i=1,2$ or 3)
β	is the ratio of the mean diameter or width of the brace members, to that of the chord

γ	is the ratio of the chord width or diameter to twice its wall thickness
Q_f	chord stress factor
n	chord stress parameter
C_f	material factor
η	is the ratio of the brace member depth to the chord diameter or width
$M_{ip,i,Ed}$	design in-plane internal moment in member i ($i=0,1,2$ or 3)
$M_{op,i,Ed}$	design out-of-plane internal moment in member i ($i = 0, 1, 2$ or 3)
$N_{i,Ed}$	design internal axial force in member i ($i=0,1,2$ or 3);
$N_{i,Rd}$	design resistance of the joint, expressed in terms of the internal axial force in member i ($i = 0, 1, 2$ or 3)

5.2. Design forces

$$N_{0,Ed} = -339\,700\,N$$

$$N_{1,Ed} = -132\,300\,N$$

$$M_{ip,0,Ed} = 0,0\,Nmm$$

$$M_{ip,1,Ed} = 0,0\,Nmm$$

$$M_{op,0,Ed} = 0,0\,Nmm$$

$$M_{op,1,Ed} = 0,0\,Nmm$$

$$N_{2,Ed} = 59\,000\,N$$

$$N_{3,Ed} = 0,0\,N$$

$$M_{ip,2,Ed} = 0,0\,Nmm$$

$$M_{ip,3,Ed} = 0,0\,Nmm$$

$$M_{op,2,Ed} = 0,0\,Nmm$$

$$M_{op,3,Ed} = 0,0\,Nmm$$

5.3. Material

$$f_{y,0} = 355,0\,MPa$$

$$f_{y,1} = 355,0\,MPa$$

$$f_{y,2} = 355,0\,MPa$$

5.4. Geometry of sections

$$A_0 = 1\,670\,mm^2$$

$$W_{el,0} = 44,4 * 10^3\,mm^3$$

5.5. Range of validity

$$0,1 + 0,01 * \frac{b_0}{t_0} = 0,1 + 0,01 * \frac{90,0}{5,0} = 0,28 \leq \frac{b_1}{b_0} = \frac{70,0}{90,0} = 0,77$$

$$0,1 + 0,01 * \frac{b_0}{t_0} = 0,1 + 0,01 * \frac{90,0}{5,0} = 0,28 \leq \frac{b_2}{b_0} = \frac{40,0}{90,0} = 0,44$$

$$0,1 + 0,01 * \frac{b_0}{t_0} = 0,1 + 0,01 * \frac{90,0}{5,0} = 0,28 \leq \frac{b_3}{b_0} = \frac{40,0}{90,0} = 0,44$$

$$0,25 \leq \frac{b_1}{b_0} = \frac{70,0}{90,0} = 0,77$$

$$0,25 \leq \frac{b_2}{b_0} = \frac{40,0}{90,0} = 0,44$$

$$0,25 \leq \frac{b_3}{b_0} = \frac{40,0}{90,0} = 0,44$$

$$0,5 \leq \frac{h_1}{b_1} = \frac{70,0}{70,0} = 1,00 \leq 2,0$$

$$0,5 \leq \frac{h_2}{b_2} = \frac{40,0}{40,0} = 1,00 \leq 2,0$$

$$0,5 \leq \frac{h_3}{b_3} = \frac{40,0}{40,0} = 1,00 \leq 2,0$$

$$\frac{b_0}{t_0} = \frac{90,0}{5,0} = 18,00 \leq 35$$

$$\frac{h_0}{t_0} = \frac{90,0}{5,0} = 18,00 \leq 35$$

$$\frac{b_1}{t_1} = \frac{70,0}{3,2} = 21,87 \leq 35$$

$$\frac{b_2}{t_2} = \frac{40,0}{2,9} = 13,79 \leq 35$$

$$\frac{b_3}{t_3} = \frac{40,0}{2,9} = 13,79 \leq 35$$

$$\frac{h_1}{t_1} = \frac{70,0}{3,2} = 21,87 \leq 35$$

$$\frac{h_2}{t_2} = \frac{40,0}{2,9} = 13,79 \leq 35$$

$$\frac{h_3}{t_3} = \frac{40,0}{2,9} = 13,79 \leq 35$$

$$t_1 = 3,2 \text{ mm} \leq t_0 = 5,0 \text{ mm}$$

$$t_2 = 2,9 \text{ mm} \leq t_0 = 5,0 \text{ mm}$$

$$t_3 = 2,9 \text{ mm} \leq t_0 = 5,0 \text{ mm}$$

$$\theta_1 = 32,1^\circ \geq 30^\circ$$

$$\theta_2 = 37,9^\circ \geq 30^\circ$$

$$\theta_3 = 87,1^\circ \geq 30^\circ$$

$$0,5 * (1 - \beta) = 0,5 * (1 - 0,61) = 0,20 \leq \frac{g}{b_0} = \frac{31,2}{90,0} = 0,34 \leq 1,5 * (1 - \beta) \\ = 1,5 * (1 - 0,61) = 0,58$$

$$g = 31,2 \text{ mm} \geq t_1 + t_2 = 3,2 + 2,9 = 6,1 \text{ mm}$$

5.6. Geometric ratios

$$\gamma = \frac{b_0}{2 * t_0} = \frac{90,0}{2 * 5,0} = 9,00$$

$$\beta = \frac{b_1 + b_2 + h_1 + h_2}{4 * b_0} = \frac{70,0 + 40,0 + 70,0 + 40,0}{4 * 90,0} = 0,61$$

5.7. Design of axial resistance of the joint

$$n = \frac{N_{o,Ed}}{A_0 * f_{y,o}} + \frac{M_{ip,0,Ed}}{W_{el,0} * f_{y,o}} + \frac{M_{op,0,Ed}}{W_{el,0} * f_{y,o}} \\ = \frac{-339\,700}{1\,670 * 355,00} + \frac{0,0}{44,4 * 10^3 * 355,00} + \frac{0,0}{44,4 * 10^3 * 355,00} = -0,57$$

C_1	
$n < 0$ (compression)	$n \geq 0$ (tension)
$C_1 = 0,50 - 0,50 * \beta$ ale $C_1 \geq 0,10$	$C_1 = 0,10$

Tab. 5.1 – Exponent for a chord stress factor

$$C_1 = 0,50 - 0,50 * \beta = 0,50 - 0,50 * 0,61 = 0,19 \geq 0,10$$

$$Q_f = (1 - |n|)^{C_1} = (1 - |-0,57|)^{0,19} = 0,85$$

C_f	
$f_y \leq 355 \text{ MPa}$	$C_f = 1,0$
$355 \text{ MPa} < f_y \leq 460 \text{ MPa}$	$C_f = 0,9$
$460 \text{ MPa} < f_y \leq 700 \text{ MPa}$	$C_f = 0,8$

Tab. 5.2 – Material factors to resistance

$$C_f = 1,0$$

$$\beta = 0,61 \leq 0,85$$

$$0,6 \leq \frac{b_1 + b_2}{2 * b_1} = \frac{70,0 + 40,0}{2 * 70,0} = 0,79 \leq 1,3$$

5.7.1. Chord face failure

$$N_{1,Rd} = \frac{8,9 * C_f * \beta * \gamma^{0,5} * \frac{f_{y0} * t_0^2}{\sin \theta_1} * Q_f}{\gamma_{M5}} = \frac{8,9 * 1,0 * 0,61 * 9,0^{0,5} * \frac{355,0 * 5,0^2}{\sin 32,1} * 0,85}{1,25}$$

$$= \frac{231\,210}{1,25} = 184\,968 \text{ N}$$

$$N_{2,Rd} = \frac{8,9 * C_f * \beta * \gamma^{0,5} * \frac{f_{y0} * t_0^2}{\sin \theta_2} * Q_f}{\gamma_{M5}} = \frac{8,9 * 1,0 * 0,61 * 9,0^{0,5} * \frac{355,0 * 5,0^2}{\sin 37,9} * 0,85}{1,25}$$

$$= \frac{200\,013}{1,25} = 160\,010 \text{ N}$$

5.8. Design check

$$\frac{N_{1,Ed}}{N_{1,Rd}} = \frac{132\,300}{184\,968} = 0,72 \leq 1,0$$

$$\frac{N_{2,Ed}}{N_{2,Rd}} = \frac{59\,000}{160\,010} = 0,37 \leq 1,0$$

It is satisfied.